

Lecture 5 - Lagrangian Techniques

Last time, developed Lagrangian formalism

$$L = T - V = \text{function of generalized coordinates } q_r$$

$$\text{then } \frac{d}{dt} \frac{\partial L}{\partial \dot{q}_r} - \frac{\partial L}{\partial q_r} = 0 \quad \text{for each } r$$

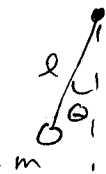
Provides equations of motion without explicitly including constraint forces

Often convenient to use polar (r, θ)
or spherical (r, φ, ϕ) coords

$$\begin{aligned} \text{For reference: } T &= \frac{1}{2} m (\dot{r}^2 + r^2 \dot{\theta}^2) && \text{polar} \\ &= \frac{1}{2} m (\dot{r}^2 + r^2 \dot{\varphi}^2 + r^2 \dot{\theta}^2 \sin^2 \varphi) && \text{spherical} \end{aligned}$$

not bad things to memorize

Example: pendulum



$$r = l \text{ fixed, so } T = \frac{1}{2} m l^2 \dot{\theta}^2$$

$$V = -mgl \cos \theta$$

$$\frac{d}{dt} \frac{\partial L}{\partial \dot{\theta}} = m l^2 \ddot{\theta}$$

$$\frac{\partial L}{\partial \theta} = -\frac{\partial V}{\partial \theta} = -mgl \sin \theta$$

$$\text{Equation of motion } \ddot{\theta} + \omega^2 \sin \theta = 0 \quad \omega^2 = \frac{g}{l}$$

Even such a simple system has no general analytic solution

If real solution needed, solve numerically

But can get analytic results for motion near equilibrium

Here equilibrium solution $\phi = 0$

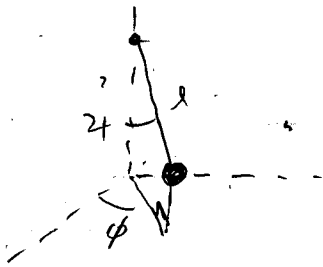
Suppose ϕ close to zero:

then $\sin \phi \approx \phi$, get

$$\ddot{\phi} + \omega^2 \phi = 0$$

Simple harmonic oscillator,
solvable

More complex example: space pendulum



$$\text{Here } T = \frac{1}{2} m l^2 (\dot{\theta}^2 + \dot{\phi}^2 \sin^2 \theta)$$

$$V = -m g l \cos \theta$$

$$\frac{\partial L}{\partial \dot{\theta}} = m l^2 \dot{\theta}$$

$$\frac{\partial L}{\partial \dot{\phi}} = m l^2 \dot{\phi} \sin^2 \theta \cos \theta - m g l \sin \theta$$

$$\text{So } \ddot{\theta} - \dot{\phi}^2 \sin \theta \cos \theta + \omega_0^2 \sin \theta = 0$$

$$\omega_0^2 = \frac{g}{l}$$

$$\frac{\partial L}{\partial \phi} = m l^2 \dot{\phi} \sin^2 \theta$$

$$\frac{\partial L}{\partial \phi} = 0$$

$$\text{So } \frac{d}{dt} (\dot{\phi} \sin^2 \theta) = 0$$

$$\dot{\phi} \sin^2 \theta = b = \text{const}$$

Again, can't solve. But can find equilibrium $\ddot{\varphi} = 0$;

$$\text{Use } \dot{\varphi} = \frac{b}{\sin^2 \varphi}$$

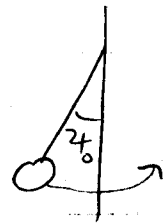
$$- \left(\frac{b}{\sin^2 \varphi} \right)^2 \sin \varphi \cos \varphi + \omega_0^2 \sin \varphi = 0$$

Either $\sin \varphi = 0$ (boring) or $b^2 \cos \varphi = \omega_0^2 \sin^4 \varphi$ (interesting)

Clear that get set. solution $\varphi = \varphi_0$ const

Mass executes uniform circular motion

Rotates at $\dot{\varphi} = \frac{b}{\sin^2 \varphi_0} = \Omega$
also constant



Can solve for φ_0 in terms of Ω :

$$- \Omega^2 \sin \varphi_0 \cos \varphi_0 + \omega_0^2 \sin \varphi_0 = 0$$

$$\cos \varphi_0 = \frac{\omega_0^2}{\Omega^2} = \frac{g}{l \Omega^2}$$

Now suppose motion near equilibrium:

Expand equations for $\varphi = \varphi_0 + \delta$

$$\text{Have } \dot{\varphi} \sin^2 \varphi = b = \Omega \sin^2 \varphi_0$$

$$\text{So } \ddot{\varphi} - \left(\Omega^2 \frac{\sin^4 \varphi_0}{\sin^4 \varphi} \right) \sin \varphi \cos \varphi + \omega_0^2 \sin \varphi = 0$$

$\varphi = \varphi_0 + \delta$, expand to first order!

$$\sin \theta \rightarrow \sin \theta_0 + \delta \cos \theta_0$$

$$\cos \theta \rightarrow \cos \theta_0 - \delta \sin \theta_0$$

$$\frac{1}{\sin^3 \theta} \rightarrow \frac{1}{\sin^3 \theta_0} \left(1 - 3\delta \frac{\cos \theta_0}{\sin \theta_0} \right)$$

Get

$$\begin{aligned} \ddot{\delta} - \Omega^2 \left(1 - 3\delta \frac{\cos \theta_0}{\sin \theta_0} \right) \sin \theta_0 (\cos \theta_0 - \delta \sin \theta_0) \\ + \omega_0^2 (\sin \theta_0 + \delta \cos \theta_0) = 0 \end{aligned}$$

Zero order terms:

$$-\Omega^2 \sin \theta_0 \cos \theta_0 + \omega_0^2 \sin \theta_0 = 0$$

$$\Rightarrow \cos \theta_0 = \frac{\omega_0^2}{\Omega^2} \quad \checkmark$$

First order:

$$\ddot{\delta} + 3\Omega^2 \delta \cos^2 \theta_0 + \Omega^2 \delta \sin^2 \theta_0 + \omega_0^2 \delta \cos \theta_0 = 0$$

$$\dot{\delta} + 2\Omega^2 \cos^2 \theta_0 \dot{\delta} + \Omega^3 \delta + \omega_0^2 \delta \cos \theta_0 = 0$$

$$\ddot{\delta} + 2\Omega^2 \frac{\omega_0^4}{\Omega^4} \delta + \Omega^2 \delta + \omega_0^2 \delta \frac{\omega_0^2}{\Omega^2} = 0$$

$$\ddot{\delta} + \left(3 \frac{\omega_0^4}{\Omega^2} + \Omega^2 \right) \delta = 0$$

Harmonic oscillator equation

$$\text{Freq } \omega^2 = 3 \frac{\omega_0^4}{\Omega^2} + \Omega^2 = \omega_0^2 \left(3 \cos^2 \theta_0 + \frac{1}{\cos^2 \theta_0} \right)$$

mass bobbles at this frequency as it swings around

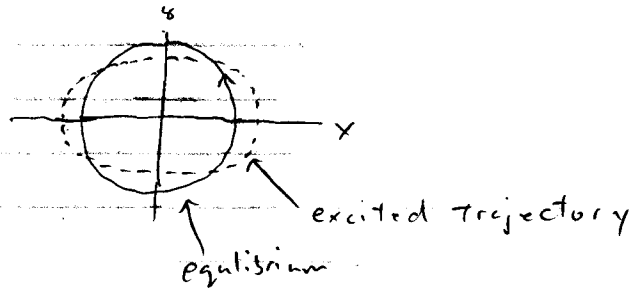


Interesting point:

For $\gamma_0 \rightarrow 0$, $\omega \rightarrow 2\omega_0 = 2\sqrt{\frac{g}{2}}$... surprising

Do get $\omega = \sqrt{\frac{3}{2}}$ if we take $\dot{\phi} = 0$ from beginning

But here still have small 3D amplitude:



See that deviation oscillates 2x faster than rotation rate

You get to do several problem like this in HW

Now, say a bit more about constraint forces
(tension, normal force, etc)

Virtue of Lagrangian method is that we don't worry about them.

But what if we want to know?

Example: how fast can we swing pendulum before string breaks?

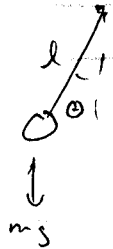
Often, this is easy to answer by elementary methods:

Know $q_i(t)$ - generalized coords
use to get $x_i(t)$ - Cartesian coords

Get $m_i \ddot{x}_i = F_i = \text{force}$

Knowing total force, usually can identify portion from constraints

Example: pendulum



Given $\theta(t)$, know

$$\begin{aligned} \text{centripetal } a_c &= l \dot{\theta}^2 \\ \text{tangential } a_t &= l \ddot{\theta} \end{aligned}$$

$$m a_c = T - mg \cos \theta$$

$$m a_t = mg \sin \theta$$

$$\text{So } T = m l \dot{\theta}^2 + mg \cos \theta$$

Easy enough

But in complicated problem, untangling a particular constraint force can be hard

Have a general approach:

Hamilton's principle tells us

$$\int_{t_1}^{t_2} \left[\sum_{\sigma} \delta q_{\sigma} \left(\frac{\partial L}{\partial q_{\sigma}} - \frac{d}{dt} \frac{\partial L}{\partial \dot{q}_{\sigma}} \right) \right] dt = 0$$

Suppose we ignored one (or more) constraints when setting up coords

For instance, kept $r (=l)$ as free coord in pendulum

Then we also have constraints $f(q, t) = 0$

$$\text{For instance, } r - l = 0$$

Then we need to minimize action,
subject to constraints

Do using Lagrange multipliers

Recall from calculus: minimize $\phi(x, y)$
subject to $f(x, y) = 0$

Make new function $\mathcal{L}(x, y, \lambda) = \phi - \lambda f$

Minimize \mathcal{L} w/ respect to x, y , and λ

Gives constrained minimum of ϕ

Don't see proof very often... I'll post one on
web page.

Here generalize to functionals:

Have $f(q_0) = 0$

If we vary path $q_0 + \delta q_0$ such that constraint
still satisfied,

$$\text{need } \delta f = \sum_i \frac{\partial f}{\partial q_{0i}} \delta q_{0i} = 0$$

Multiply by arbitrary functions $\lambda(q, \dot{q}, t)$, still zero

Add to integrand; still zero:

$$\int \sum_i \delta q_{0i} \left[\frac{\partial L}{\partial q_{0i}} - \frac{d}{dt} \frac{\partial L}{\partial \dot{q}_{0i}} + \sum_j \lambda_j \frac{\partial f_j}{\partial q_{0i}} \right] dt = 0$$

↑
one term for each
additional constraint

If we have n coords and k constraints,
really only $n-k$ degrees of freedom

\Rightarrow different q_{σ} 's are not independent

So we can demand that $[]$ terms vanish
for $\sigma = 1$ to $n-k$ only

But we can now choose λ_k 's such that
 $[]$ for $\sigma = n-k+1$ to n vanish as well

End up with

$$\frac{d}{dt} \frac{\partial L}{\partial \dot{q}_{\sigma}} - \frac{\partial L}{\partial q_{\sigma}} - \sum_{j=1}^k \lambda_j \frac{\partial f_j}{\partial q_{\sigma}} = 0 \quad \sigma = 1 \text{ to } n$$

and $f_j(q) = 0 \quad j = 1 \text{ to } k$

Total of $n+k$ equations

Have $n+k$ unknowns: $\{q_{\sigma}\}$ and $\{\lambda_j\}$

So we can solve for q 's & λ 's: gives constrained solution

Why do all this?

Can see physical significance of λ :

Suppose we instead included constraint forces
directly in L

Then would have constraint potential V_c i.e.

$$L_2 \Rightarrow T - V - V_c = L^{(0)} - V_c(q_s)$$

Get equation of motion

$$\frac{d}{dt} \frac{\partial L^{(0)}}{\partial \dot{q}_\sigma} - \frac{\partial L^{(0)}}{\partial q_\sigma} + \frac{\partial U_c}{\partial q_\sigma} = 0$$

See that term $\lambda \frac{\partial f}{\partial q_\sigma}$ corresponds to $-\frac{\partial U_c}{\partial q_\sigma}$

= "generalized" constraint force Q_σ

or,
$$\lambda_j \frac{\partial f_j}{\partial q_\sigma} \equiv Q_\sigma^{(j)}$$

So, knowing λ lets you get constraint forces

Can relate to proper force F_σ by work:

If motion δq_σ violates constraint, then constraint force does work

$$\delta W = Q_\sigma \delta q_\sigma = \sum_i F_i \delta x_i$$

Given x_i (q_σ 's), can solve for F_i 's as desired

Usually straight forward

Example: pendulum

Set up with r, θ both free

$$L = \frac{1}{2} m (\dot{r}^2 + r^2 \dot{\theta}^2) + mgr \cos \theta$$

Constraint $f(r, \theta) = r - l = 0$

$$\frac{\partial f}{\partial r} = 1 \quad \frac{\partial f}{\partial \theta} = 0$$

Get equations of motion:

$$(1) \quad m(\ddot{r} - r\dot{\theta}^2) - mg \cos \theta = \lambda$$

$$(2) \quad \frac{d}{dt}(mr^2\dot{\theta}) + mgr \sin \theta = 0$$

$$(3) \quad r = l$$

In solution, immediately have $\dot{r} = \ddot{r} = 0$

$$(1) \rightarrow -ml\dot{\theta}^2 - mg \cos \theta = \lambda$$

and

$$(2) \rightarrow \underbrace{ml^2\ddot{\theta} + mgl \sin \theta = 0}_{\hookrightarrow \ddot{\theta} + \omega^2 \sin \theta = 0}$$

as before

Given solution, have $\lambda = \lambda \frac{\partial f}{\partial r} = Q_r$

$$\text{so } Q_r = -ml\dot{\theta}^2 - mg \cos \theta$$

If we tried to increase r , would do work

$$\delta W = Q_r \delta r = -T \delta r$$

↑ tension directed inward,
but δr outward

$$\text{So tension } T = -Q_r \\ = ml\dot{\theta}^2 + mg \cos \theta$$

As before. Such a simple problem doesn't merit complicated formalism, but we'll see more complicated example next time